

## SIDDARTHA INSTITUTE OF SCIENCE AND TECHNOLOGY :: PUTTUR (AUTONOMOUS)

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#### **QUESTION BANK (DESCRIPTIVE)**

Subject with Code: R & AC (18ME0336)Course & Branch: B.Tech -MERegulation: R18Year & Sem: IV-B.Tech & I-Sem

#### <u>UNIT –I</u> Introduction

	introduction									
1	a	Define the term Refrigeration.	[L1]	[C01]	[2M]					
	b	Define the term Heat Pump.	[L1]	[C01]	[2M]					
	с	Define C.O.P.	[L1]	[C01]	[2M]					
	d	State any four applications of refrigeration.	[L3]	[C01]	[2M]					
	e	Draw P-V and T-S diagram of Bell-Coleman cycle.	[L1]	[C01]	[2M]					
2	a	Define Unit of Refrigeration.	[L1]	[C01]	[4M]					
	b	Explain the working of a Reversed Carnot cycle of refrigeration with P-V and T-S Diagrams.	[L5]	[C01]	[6M]					
3		With neat sketch Explain the working of Simple air refrigeration system	[L1]	[C01]	[10M]					
4	a	What are the limitations of Carnot cycle of refrigeration?	[L1]	[C01]	[5M]					
	b	Describe Boot strap air refrigeration system, with a schematic diagram and show the cycle on T-S Diagram.	[L1]	[C01]	[5M]					
5	a	State the applications of refrigeration.	[L3]	[C01]	[5M]					
	b	Explain, with a neat sketch the working principle of Regenerative Air refrigeration system.	[L5]	[C01]	[5M]					
6	a	What is the Necessity of refrigeration?	[L1]	[C01]	[5M]					
	b	Describe with a neat sketch a Reduced ambient air refrigeration system	[L1]	[C01]	[5M]					
7		In a refrigeration plant working on Bell Coleman cycle, air is compressed to 5 bar from 1 bar. Its initial temperature is $10^{\circ}$ C. After compression, the air is cooled up to $20^{\circ}$ C in a cooler before expanding to a pressure of 1 bar. Determine the theoretical C.O.P of the plant and net refrigerating effect. Take Cp = $1.005$ KJ/Kg K and Cv = $0.718$ KJ/Kg K.	[L5]	[C01]	[10M]					
8		A refrigerator working on Bell Coleman cycle operates between pressure limits of 1.05 bar and 8.5 bar. Air is drawn from the cold chamber at 10 °C, compressed and then it is cooled to 30 °C before entering the expansion cylinder. The expansion and compression	[L5]	[C01]	[10M]					

	follows the law PV <sup>1.3</sup> = constant. Determine the theoretical C.O.P of			
	the system.			
9	An air refrigerator working on Bell Coleman cycle takes the air into the compressor at 1 bar and -7 °C and is compressed isentropically to 5.5 bar and it is further cooled to 18 °C at the same pressure. Find the C.O.P of the system if (a). The expression is isentropic (b). The expression follows the law PV $^{1.25}$ = constant. Take $\gamma$ = 1.4 and Cp = 1 KJ/Kg K.	[L4]	[C01]	[10M]
10	An air refrigerator used for food storage provides 50 tons of refrigeration. The temperature of air entering the compressor is 7 ° C and the temperature before entering into expander is 27 ° C .Assuming 30 % more power is required than theoretical, find (a).Actual C.O.P of the cycle (b).KW capacity required to run the compressor.	[L5]	[C01]	[10M]

#### <u>UNIT –II</u> <u>Vapour Compression Refrigeration System</u>

1	a	Draw P-H di	agram of VC	CR cycle.					[L1]	[C02]	[2M]
	b	State purpos	e of accumul	lator in VCR	system.				[L5]	[C02]	[2M]
	c	Mention the	uses of flash	chamber.					[L2]	[C02]	[2M]
	d	Draw T-S of	VCR cycle.						[L1]	[C02]	[2M]
	e	What are the functions of compressor in vapour compression						[L1]	[C02]	[2M]]	
		refrigeration system?									
2	a	State the functions of expansion device.						[L1]	[C02]	[5M]	
	b	Construct P	ressure – En	thalpy (p-h)	chart of Vap	or com	pressi	on cycle	[L6]	[C02]	[5M]
3		The tempera	ture limits o	of an ammon	ia refrigerat	ing sys	stem a	re 25° C	[L5]	[C02]	[10M]
		and -10 $^{\circ}$ C.	If the gas is	dry at the	end of comp	ression	, calc	ulate the			
		coefficient of performance of the cycle assuming no under cooling of									
		the liquid ammonia. Use the following table for properties of									
		ammonia.									
		Temperatur	re • Liqui	id Heat	Latent Hea	t	Liqui	id Entropy			
		С	(Kj	/ kg)	(Kj/kg)			kg K)			
		25		298.9		1166.94 1.1242					
		-10		135.37	1297.6						
4		A Vapour co	-	•		-	-		[L5]	[C02]	[10M]
		of 60 bar a		-		-					
		compression and there is no under cooling of the liquid before the									
		expansion valve. Determine (i). C.O.P of the cycle (ii). Capacity of the									
		refrigerator if the fluid flow is at the rate of 5 kg/min.									
		Pressure	Temperat	Enthalpy (k	j / kg)	Entro	ру (К	(j/kgK)			
		(Bar)	ure o C	Liquid	Vapour	Liqui		Vapour			
		60	295	151.96	293.29	0.5		1.0332			
		25	261	56.32	322.58	0.2	26	1.2464			

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5		28 tonnes ice	from and at 0	) ° C is produc	ed per	day in a	n ammonia	[L5]	[C02]	[10M]
				e range in the co	_	=		[]	[ E = \ ]	[
		_	-	d saturated at t	-					
			=	There is no liq		_				
		_		theoretical, Ca	="	_	_			
				lowing properti		_	=			
		Temperatur	Enthalpy (kj			opy (Kj/				
		e o C	Liquid	Vapour	Liqu	15 \ 5	Vapour			
		25	298.9	1465.84	1.	.1242	5.0391			
		-15	112.34	1426.54	0.	.4572	5.5490			
6	<u> </u>	A refrigeratio	n machine usi	ing R-12 as re	frigera	nt operate	es between	[L6]	[C02]	[10M]
		_		bar. The con	-		-			
			_	n the condens		•	•			
				beginning of		-				
				ctual C.O.P is						
				oduced per hou	r. The	refrigerar	nt flow is 5			
			erties of refriger							
			-	Inthalpy (kj / kg	-		of saturated			
		$\left \begin{array}{c} e \\ O \end{array}\right  \circ O$	$\mathcal{L}$	iquid Vap	oour	vapour, l	kj / kg K			
		(Bar) 9	36	70.55 20	01.8	<del>                                     </del>	0.6836			
		2.5	-7		84.5		0.7001			
7	a			ome examples t				[L1]	[C02]	[5M]
	b						_	[L1]	[C02]	[5M]
		What are the advantages of vapour compression refrigeration system [L1] over air refrigeration system?								
8	a			of refrigerants	<u> </u>			[L1]	[C02]	[5M]
	b			nts generally us				[L1]	[C02]	[5M]
9				rigeration plan		s betwee	en pressure	[L4]	[C02]	[10M]
		limits of 5.3 b	par and 2.1 bar	The vapour is	super-	-heated at	t the end of			
		compression,	its temperature	e being 37 ° C.	The va	pour is sı	uper-heated			
		by 5 ° C before	e entering the c	compressor.						
		If the specific	heat of super	-heated vapour	is 0.6.	3 kj / kg	k, find the			
		coefficient of	performance of	f the plant. Use	the dat	ta gi <u>ven t</u>	pelow			
		Pressure (Bar)	) Temperature C	e • Liquid H /kg)	leat	(kj Later	nt Heat (kj/kg			
		5.3	15.5		.15		144.9			
		2.1	-14	25	.12		158.7			
10	a	Sketch and ex	plain a two-sta	ige cascade refr	igeratio	on system	1.	[L2]	[C02]	[5M]
	b		<del>-</del>	lain the work	cing p	rinciple	of vapour	[L5]	[C02]	[5M]
		compression r	refrigeration sys							
				<u>UNIT</u>						
				Other Refriger	nation (	Stratoma				

### **Other Refrigeration Systems**

R18

	b	Mention any four desirable properties of refrigerant and absorbent pair.	[L1]	[C03]	[2M]
	c	List the secondary refrigerants.	[L1]	[C04]	[2M]
	d	What are the refrigerants used in Electrolux refrigeration system	[L4]	[C04]	[2M]
	e	What are the factors to be considered while selecting a refrigeration system?	[L1]	[C04]	[2M]
2	a	Discuss properties of refrigerant and absorbent combination used in vapour absorption system	[L4]	[C03]	[5M]
	b	State the advantages and limitations of Vapour absorption refrigeration system	[L1]	[C03]	[5M]
3		Explain with a neat sketch the working of lithium-bromide vapour absorption system.	[L2]	[C03]	[10M]
4		Explain with help of a neat sketch, the working of a steam jet refrigeration system.	[L2]	[C03]	[10M]
5	a	Comparison between two fluid VAR system and three fluid VAR system.	[L4]	[C03]	[5M]
	b	Define the terms nozzle efficiency and entrainment efficiency in steam jet refrigeration system.	[L1]	[C03]	[5M]
6	a	Illustrate the working principal of Electrolux refrigeration system	[L2]	[C04]	[5M]
	b	Advantages of vapour absorption refrigeration system over vapour compression refrigeration system	[L5]	[C04]	[5M]
7		Differentiate between vapour absorption and vapour compression refrigeration systems.	[L4]	[C04]	[10M]
8		Describe the working of a vapour absorption refrigeration system with the help of a neat sketch.	[L1]	[C04]	[10M]
9		Explain thermo-electric refrigeration system with sketch	[L2]	[C04]	[10M]
10		Describe the working of Vortex tube with a neat sketch and its merits and demerits	[L1]	[C04]	[10M]

#### <u>UNIT –IV</u> <u>Introduction to Air Conditioning</u>

1	a	Define Dew point Depression.	[L1]	[C05]	[2M]
	b	Define the term effective temperature.	[L1]	[C05]	[2M]
	c	Define term air conditioning.	[L1]	[C05]	[2M]
	d	Write the classifications of air conditioning systems.	[L1]	[C05]	[2M]
	e	Define psychrometry.	[L1]	[C05]	[2M]
2		A room $7m \times 4m \times 4m$ is occupied by an air-water vapour mixture at	[L5]	[C05]	[10M]
		38°C. The atmospheric pressure is 1 bar and the relative humidity is			
		70%. Determine the humidity ratio, dew point, mass of dry air and mass			
		of water vapour. If the mixture of air-water vapour is further cooled at			
		constant pressure until the temperature is 10°C. Find the amount of water			
		vapour condensed			

4	a b	Define Sensible heat factor.  With help of psychrometric chart, Explain the following processes (i). Sensible hearting (ii) Sensible cooling  Atmospheric air at 0.965 bar enters the adiabatic saturator. The wet bulb temperature is 20°C and dry bulb temperature is 31°C during adiabatic saturation process. Determine (i) humidity ratio of the entering air (ii) vapour pressure and relative humidity at 31°C and (iii) dew point temperature.	[L1] [L5]	[C05] [C05]	[5M] [5M]
4		(i). Sensible hearting (ii) Sensible cooling  Atmospheric air at 0.965 bar enters the adiabatic saturator. The wet bulb temperature is 20°C and dry bulb temperature is 31°C during adiabatic saturation process. Determine (i) humidity ratio of the entering air (ii) vapour pressure and relative humidity at 31°C and (iii) dew point			
4	a	Atmospheric air at 0.965 bar enters the adiabatic saturator. The wet bulb temperature is 20°C and dry bulb temperature is 31°C during adiabatic saturation process. Determine (i) humidity ratio of the entering air (ii) vapour pressure and relative humidity at 31°C and (iii) dew point	[L5]	[C05]	[10M]
·	a	temperature is 20°C and dry bulb temperature is 31°C during adiabatic saturation process. Determine (i) humidity ratio of the entering air (ii) vapour pressure and relative humidity at 31°C and (iii) dew point		[600]	
	a	saturation process. Determine (i) humidity ratio of the entering air (ii) vapour pressure and relative humidity at 31°C and (iii) dew point			
	a	vapour pressure and relative humidity at 31°C and (iii) dew point			
	a				1
	a	temperature.			
5	и	With help of psychrometric chart, Explain the Heating and	[L5]	[C05]	[5M]
5		dehumidification processes	[]	[]	[e]
	b	With help of psychrometric chart, Explain the cooling and	[L5]	[C05]	[5M]
	Ü	humidification processes	[]	[]	[e]
6	a	Define relative humidity, absolute humidity.	[L1]	[C05]	[5M]
	b	Define saturated air, degree of saturation.	[L1]	[C05]	[5M]
7	a	Explain the procedure to draw a grand sensible heat factor line on a	[L5]	[C05]	[5M]
·		psychrometric chart.	. ,		. ,
	b	What do you understand by the term psychrometry?	[L1]	[C05]	[5M]
8	a	Explain the concept of effective room sensible heat factor with neat	[L5]	[C05]	[5M]
O		diagram.	1	[ ]	r j
	b	Define the following (i). Specific humidity (ii). Absolute Humidity	[L1]	[C05]	[5M]
9		A room has a sensible heat gain of 24 KW and a latent heat gain of 5.2	[L5]	[C05]	[10M]
		KW and it has to be maintained at 26 ° C DBT and 50 % RH.180 m <sup>3</sup> /			
10		min of air is delivered to the room. Determine the state of supply of air.	FT 12	[C05]	[10](1
10		Define the following terms (i)Infiltration (ii)Natural ventilation (iii)	[L1]	[C05]	[10M]
		Forced ventilation			
		UNIT –V			

# <u>UNIT -V</u> <u>Air Conditioning Systems and Distribution of Air</u> ristics of good distribution system.

1	a	List out characteristics of good distribution system.	[L2]	[C06]	[2M]
	b	What is comfort chart?	[L1]	[C06]	[2M]
	c	What are the materials used for duct.	[L1]	[C06]	[2M]
	d	Write continuity equation in ducts.	[L1]	[C06]	[2M]
	e	Define Duct.	[L1]	[C06]	[2M]
2		Explain winter air conditioning system with sketch	[L2]	[C06]	[10M]
3		With neat diagram explain the working of summer air conditioning	[L2]	[C06]	[10M]
		system.			
4	a	Explain the working of domestic refrigerator with a neat sketch.	[L2]	[C06]	[5M]
	b	Explain year round air conditioning system with sketch.	[L2]	[C06]	[5M]
5	a	Define the terms static and velocity pressure in a duct.	[L1]	[C06]	[5M]
	b	Define the term duct. Explain the needs.	[L1]	[C06]	[5M]
			&L2		
			]		

6		Compare winter air conditioning system with summer air conditioning	[L2]	[C06]	[10M]
		system.			
7	a	Derive an expression for continuity equation in ducts.	[L4]	[C06]	[5M]
	b	The main air supply duct of an air conditioning system is 800 mm X 600	[L5]	[C06]	[5M]
		mm in cross section and carries 300 m <sup>3</sup> / min of standard air. It branches			
		into two ducts of cross section 600 mm X 500 mm and 600 mm X 400			
		mm. If the mean velocity in the larger branch is 480 m / min. Find (i)			
		Mean velocity in the main duct and the smaller branch (ii) mean velocity			
		pressure in each duct.			
8		Following data refers to an air conditioning system to be designed for an	[L5]	[C06]	[10M]
		industrial process for hot and wet climate. Outside conditions 30 ° C DBT			
		and 75 % RH, Inside conditions 20 ° C DBT and 60 % RH.			
		The require condition is to be achieved first by cooling and			
		dehumidifying and then by heating. If 20 m <sup>3</sup> of air is absorbed by the			
		plant every minute. Find (i) Capacity of the cooling coil in tonnes of			
		refrigeration (ii) Capacity of the heating coil in KW (iii) Amount of water			
		removed per hour. Take $h_1=81.8$ kj/kg, $h_2=34.2$ kj/kg, $h_3=42.6$ kj/kg,			
		$W_1=0.0202 \text{ kj/kg}, W_2=0.0088 \text{ kj/kg}, V_{s1}=0.886 \text{ m}^3/\text{kg}.$			
9	a	Why the ducts are used in an air conditioning system.	[L1]	[C06]	[5M]
	b	Which material is commonly used for making ducts in air conditioning	[L1]	[C06]	[5M]
		systems?			
10		An air conditioning plant is required to supply 60 m <sup>3</sup> of air per minute at	[L5]	[C06]	[10M]
		a DBT of 21°C and 55 % RH. The outside air is at DBT of 28 ° C and 60			
		% RH. Determine the mass of water drained and capacity of the cooling			
		coil. Assume the air conditioning plant first to dehumidify and then to			
		cool the air. Take $W_1$ =0.0142, $W_2$ =0.0084 kj /kg of dry air, $V_{s2}$ =0.845 m <sup>3</sup>			
		$/ \text{ kg, h}_1 = 64.8 \text{ kj/kg, h}_2 = 42.4 \text{ kj/kg.}$			

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